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Project

Title

CHLOE @ SKY628

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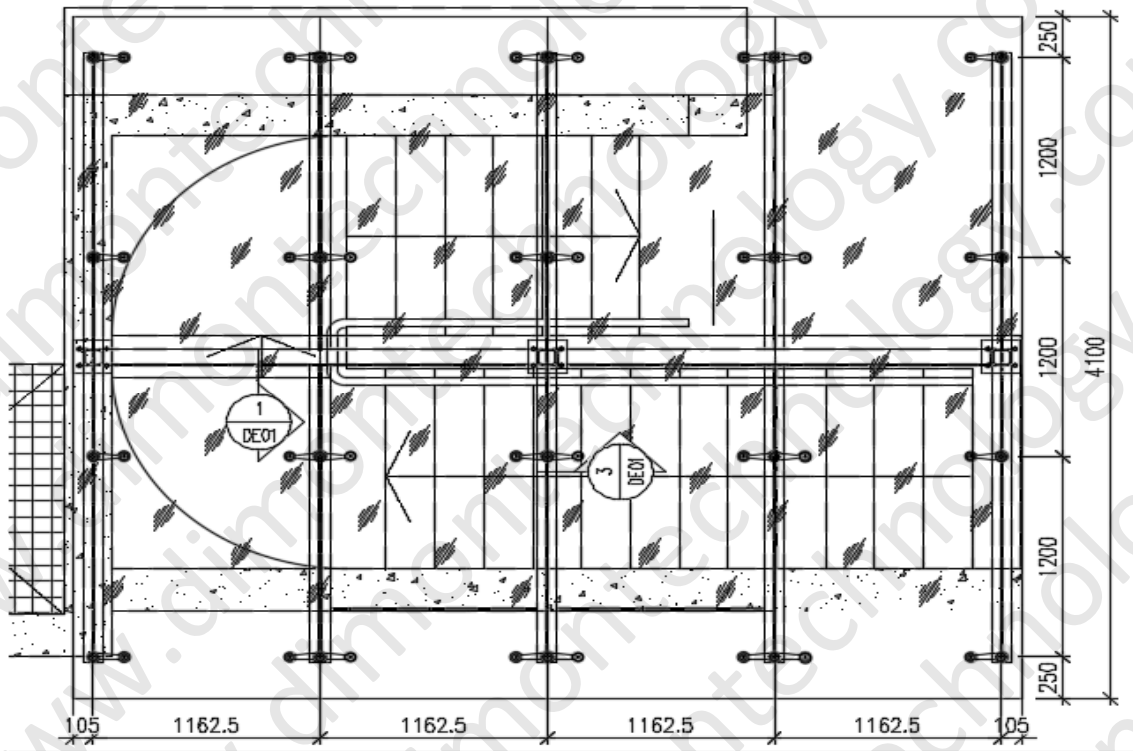
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1. Introduction

The skylights are made of 12+12mm thick laminated tempered glass fixed to s.s. RHS by means of stainless steel spider system. The objective of this calculation is to check the design of skylight to be safe against the dead load, live load and wind load.

Load path

Wind load to skylight → glass panels → s.s. spiders → steel RHS frame → gms embeds → r.c. structures



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2. Design Code, Design Data

2.1 Design Code

- Hong Kong Building (Construction) Regulation 1990 (Amendment 2011).
- Code of Practice on Wind Effect 2008, Macau.
- Code of Practice for the Structural Use of Steel 2011, Hong Kong
- Code of Practice for the Structural Use of Concrete 2004, Hong Kong
- Code of Practice for Dead and imposed Loads 2011, Hong Kong
- Code of Practice for the Structural Use of Steel GB 50017-2003
- Load Code of the design of buliding structure GB 50009-2012

2.2 Design Data

2.2.1 Stainless steel to be grade X5CrNiMo17-12-2 complied with BS EN 10088

stainless steel grade		1.4401 (316 S31) X5CrNiMo17-12-2
0.2% proof stress	(N/mm ²)	220
ultimate tensile strength, min.	(N/mm ²)	510
Modulus of elasticity	(N/mm ²)	200000
Design strength	(N/mm ²)	220
Design stress of fillet weld	(N/mm ²)	220

2.2.2 All welding to be complied with complied to GB50661-2011

steel grade		E50xx
design strength of filled weld	(N/mm ²)	200

2.2.3 All stainless steel bolts or screws to be grade A4 complied to BS EN ISO 3506

class		70
0.2% proof stress	(N/mm ²)	450
ultimate tensile strength	(N/mm ²)	700
design tensile strength	(N/mm ²)	373
design shear strength	(N/mm ²)	280
design bearing strength	(N/mm ²)	805
stainless steel grade		A4 / 316
design bearing strength on connected part	(N/mm ²)	479

2.2.4 Concrete

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concrete strength,	f_{cu} (N/mm ²)	45	
anchorage bond strength, ($= \beta f_{cu}^{0.5}$)	f_{bu} (N/mm ²)	1.878	$\beta = 0.28$ for cast-in threaded rod
design concrete shear stress,	v_c (N/mm ²)	0.4	
compressive strength, ($= 0.6 f_{cu}$)	(N/mm ²)	27	

2.2.5 The date refer to SAP2000 program.

3. Design Load

3.1 Wind load

Wind pressure, $q_z = 1.84$ kPa (height above ground level $\leq 10m$)

Pressure coefficient, $= 2$ (open frame)

Design wind load
 $= 2.12 \times 2.0$
 $= 3.68$ kPa

3.2 Live load

Live load $= 0.75$ kN/m²

3.3 Dead Load

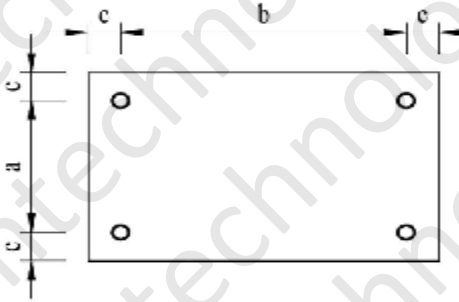
Weight of glass $= (12 + 12) \times 26.5 / 1000 = 0.636$ kPa

Others, $= 0.064$ kPa

Total $= 0.7$ kPa

4. Check for laminated tempered glass

4.1 Check for 12mm+1.56PVB+12mm clear laminated tempered glass



Max. bending stress, $\sigma_c = \alpha q b^2 / t^2$
 Max. deflection, $\delta_c = \beta q b^4 / (E_g t^3)$

a/b	β			α		
	b/c=10	b/c=15	b/c=20	b/c=10	b/c=15	b/c=20
1	0.2547	0.2668	0.273	0.8194	0.8719	0.8719
0.95	0.2302	0.2414	0.2472	0.8087	0.843	0.858
0.9	0.2102	0.2206	0.2259	0.7984	0.8307	0.8447
0.85	0.1934	0.203	0.2079	0.7886	0.819	0.832
0.8	0.1801	0.189	0.1935	0.7792	0.8079	0.8199
0.75	0.1693	0.1776	0.1816	0.7703	0.7974	0.8085
0.7	0.1611	0.1688	0.1724	0.762	0.7876	0.7979
0.65	0.1549	0.1619	0.1653	0.7543	0.7786	0.7881
0.6	0.1504	0.157	0.1601	0.7473	0.7703	0.7792
0.55	0.1513	0.1567	0.1593	0.741	0.7629	0.7712
0.5	0.1512	0.1565	0.1588	0.7355	0.7564	0.7641

Glass density, $\rho = 2650$ kg/m³
 Nominal thickness of glass pane 1 = 12 mm
 Nominal thickness of glass pane 2 = 12 mm
 Glass type for pane 1 & 2 : tempered
 Load duration, self weight : long term

Downward load
 Basic wind pressure, $q_z = 1.84$ kN/m²
 Topography factor, $S_a = 1$
 Pressure coefficient, $C_p = 2$
 Design wind pressure, $q_{dn} = q_z S_a C_p = 3.68$ kN/m²
 Live load, $q_k = 0.75$ kN/m²
 Glass weight = $\rho \Sigma (t / \gamma_d)$, $g_{k,dn} = 0.96$ kN/m²

Min. thickness of glass pane 1, $t_1 = 11.91$ mm
 Min. thickness of glass pane 2, $t_2 = 11.91$ mm
 Reduction factor, $\gamma_d = 0.66$

Upward load
 Basic wind pressure, $q_z = 1.84$ kN/m²
 Topography factor, $S_a = 1$
 Pressure coefficient, $C_p = -2$
 Design wind pressure, $q_{up} = q_z S_a C_p = -3.68$ kN/m²
 Glass weight = $\rho \Sigma t$, $g_{k,up} = 0.63$ kN/m²

Load combinations (downward)
 $1.0 w_{dn} + 1.0 q_k = 4.43$ kN/m²
 $1.3 g_{k,dn} + 0.9 w_{dn} + 1.05 q_k = 5.35$ kN/m²
 $1.2 g_{k,dn} + 1.5 w_{dn} + 1.05 q_k = 7.46$ kN/m²
 $1.2 g_{k,dn} + 0.9 w_{dn} + 1.5 q_k = 5.59$ kN/m²

Load combinations (upward)
 $1.0 w_{up} + 1.0 g_{k,up} = -3.05$ kN/m²
 $1.5 w_{up} + 1.0 g_{k,up} = -4.89$ kN/m²

Serviceability load, $w_{dns} = q_{dn} + q_k = 4.43$ kN/m²

Serviceability load, $w_{ups} = q_{up} + g_s = -3.05$ kN/m²

Critical combination, $w_c = 7.46$ kN/m²
 Critical serviceability, $w_{cs} = 4.43$ kN/m²

downward
 downward (for deflection checking)

Glass type for pane 1 & 2 : tempered
 Load duration : short term

Ultimate design strength, $p_y = 80$ N/mm²
 Reduction factor, $\gamma_d = 1$

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Surface treatment : clear

Reduction factor, $\gamma_s = 1$

Material factor, $\gamma_m = 1$

Ultimate resistance strength, $R_{ult} = p_y \gamma_d \gamma_s / \gamma_m = 80 \text{ N/mm}^2$

Critical load shared equally to each pane, $w_1 = 7.46 / 2 = 3.73 \text{ kN/m}^2$

Critical serviceability load shared equally to each pane, $w_{s1} = 4.43 / 2 = 2.215 \text{ kN/m}^2$

The glass panel is simply supported on 4 points.

Longer side of glass pane, $b = 1200 \text{ mm}$

Shorter side of glass pane, $a = 918 \text{ mm}$

Edge distance, $c = 125 \text{ mm}$

Modulus of Elasticity, $E_g = 70000 \text{ N/mm}^2$

By interpolation, $a/b = 0.765$ $b/c = 9.6$

$\alpha = 0.7708$

$\beta = 0.1719$

Max. bending stress of glass pane, $\sigma_{c1} = 0.7708 \times 3.73 / 1000 \times 1200^2 / 11.91^2 = 29.19 \text{ N/mm}^2 \leq 80 \text{ N/mm}^2$ O.K.

Max. deflection, $\delta_c = 0.1719 \times 2.215 / 1000 \times 1200^4 / (70000 \times 11.91^3) = 6.68 \text{ mm} \leq 1200 / 60 = 20 \text{ mm}$ O.K.

4.2 Check for flat cap routel (Kin Long TF12)

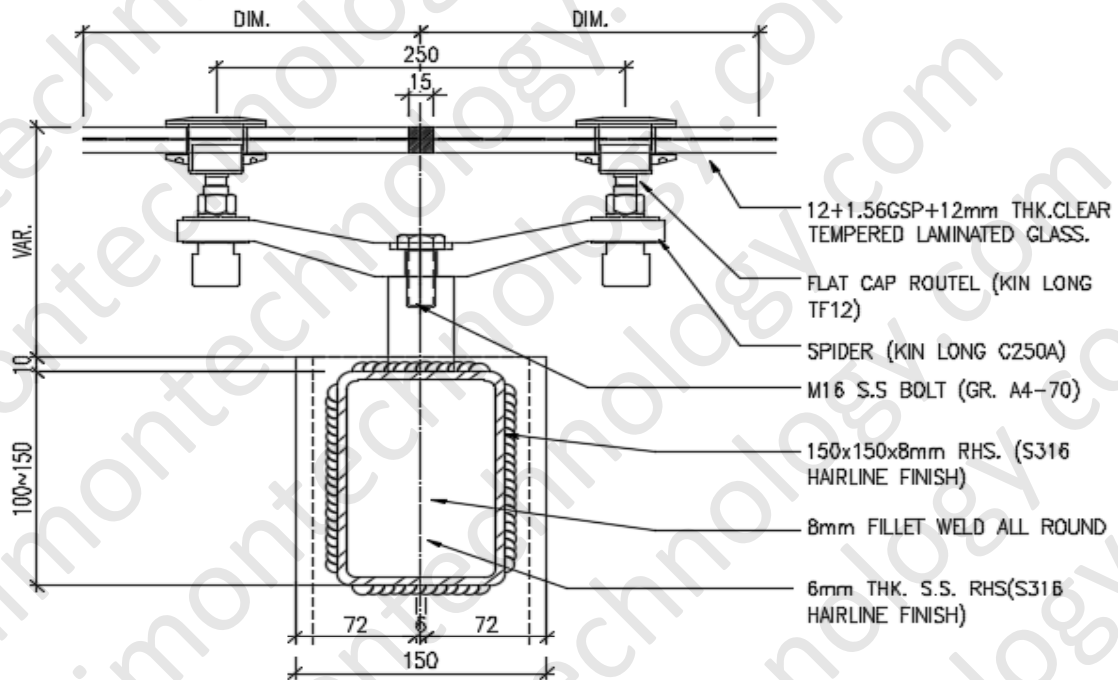
Vertical load $= 0.7 + 0.75 + 3.68 \text{ (DL+LL + WL)}$
 $= 5.13 \text{ kN/m}^2 \text{ (unfactored)}$

Vertical load on routel at centre $= 5.13 \times (1200 / 2 + 250) \times (1162.5 / 2 + 105) / 10^6 = 2.99 \text{ kN}$

Reaction on each routel point tension/compression $= 2.99 \text{ kN}$ (refer to Appendix)
 $< 4.5 \text{ kN}$ O.K.

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4.3 Check for spider (Kin Long C250A)



Vertical load on spider = 2.99 kN (refer to item 4.2)

Reaction on each spider tension/compression = 2.99 kN (refer to Appendix)
 < 4 kN O.K.

4.4 Check for M16 s.s. bolt, A4-70

Vertical load = 2 x 2.909 = 5.818 kN (refer to item 4.2)

Tensile area of M16 bolt = 156 mm²
 Tensile strength of M16 bolt = 373 N/mm²
 Shear strength of M16 bolt = 280 N/mm²

Tensile capacity of bolt = 156 x 373 / 1000 = 58.19 kN
 > 5.818 kN O.K.

Pull out capacity on connected part, = 16 x 3.1416 x 10 / 2 x 127 / 1000 = 31.919 kN
 > 5.818 kN O.K.

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Check for 5mm fillet weld connector channel to steel frame

$$\text{Horizontal load,} = 5.818 \quad \text{kN}$$

$$\begin{aligned} \text{Effective length of fillet weld,} &= 2 \times 3.14 \times 23 \\ &= 144.44 \quad \text{mm} \end{aligned}$$

$$\begin{aligned} \text{Capacity of fillet weld} &= 160 \times 144.44 \times 5 \times 0.7 / 1000 \\ &= 80.886 \quad \text{kN} \\ &> 5.818 \quad \text{kN} \end{aligned}$$

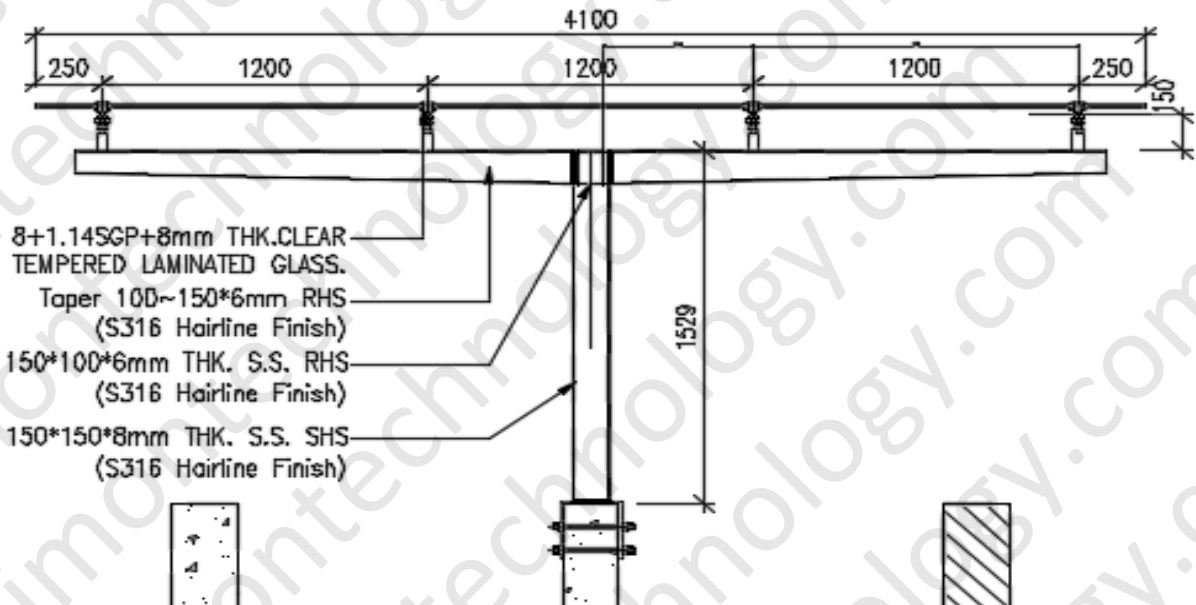
O.K.

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5. Check for steel frame



5.1 Check for 150x100x6mm RHS

$$\text{Vertical load} = 7.46 \text{ kN/m}^2$$

$$\begin{aligned} \text{Point load } P_1 &= 7.46 \times 1.2 \times 1.1625 \\ &= 10.407 \text{ kN} \end{aligned}$$

$$\begin{aligned} P_2 &= 7.46 \times (1.2 / 2 + 0.25) \times 1.1625 \\ &= 7.371 \text{ kN} \end{aligned}$$

$$\text{Length of cantilever} = 1850 \text{ mm}$$

$$\begin{aligned} \text{Reaction at support} &= 10.407 + 7.371 \\ &= 17.778 \text{ kN} \end{aligned}$$

$$\begin{aligned} \text{Max bending moment} &= 10.407 \times 0.55 + 7.371 \times 1.75 \\ &= 18.62 \text{ kNm} \end{aligned}$$

$$\begin{aligned} \text{Shear stress} &= 17.778 \times 1000 / (6 \times 150) \\ &= 19.75 \text{ N/mm}^2 \\ &< 0.6 \times 127 = 76.2 \text{ N/mm}^2 \quad (\text{low shear}) \end{aligned}$$

$$\begin{aligned} \text{Moment capacity} &= \min(1.2 p_y Z_y, p_y S_y) \\ &= (1.2 \times 220 \times 115 / 1000, 220 \times 141 / 1000) \\ &= 30.36 \text{ kNm} \quad (\text{factored}) \\ &> 18.62 \text{ kNm} \quad \text{O.K.} \end{aligned}$$

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$$\begin{aligned}
 \text{Max deflection} &= P_k b^2(3L-b) / (6E I) \\
 &= 10.407 \times 5.13 / 7.46 \times 550^2 \times (3 \times 1850 - 550) / (6 \times 200000 \times 8620000) + \\
 &\quad 7.371 \times 5.13 / 7.46 \times 1750^2 \times (3 \times 1850 - 1750) / (6 \times 200000 \times 8620000) + \\
 &= 6.75 \quad \text{mm} \quad (\text{unfactored}) \\
 &< 2 \times 1850 / 250 = 14.8 \quad \text{mm} \quad \text{O.K.}
 \end{aligned}$$

Check for lateral torsional buckling

$$\begin{aligned}
 \text{Design strength, } p_y &= 220 \quad \text{N/mm}^2 & \text{Parameter, } \epsilon &= (275 / p_y)^{0.5} = 1.118 \\
 \text{Modulus of Elasticity, } E &= 200000 \quad \text{N/mm}^2
 \end{aligned}$$

Overall width, B	= 100	mm	Overall depth, D	= 150	mm
Wall thickness, t	= 6	mm	Area, A	= 28.2	cm ²
Moment of inertia, I _x	= 862	cm ⁴	Moment of inertia, I _y	= 456	cm ⁴
Section modulus, Z _x	= 115	cm ³	Section modulus, Z _y	= 91.2	cm ³
Plastic modulus, S _x	= 141	cm ³	Plastic modulus, S _y	= 106	cm ³
Torsional constant, J	= 946	cm ⁴	Radius of gyration, r _y	= 4.02	cm
Effective length, L _E	= 2362	mm			

$$\begin{aligned}
 \text{Slenderness ratio, } \lambda &= L_E / r_y = 46 \\
 \gamma_b &= (1 - I_y / I_x) [1 - J / (2.6 I_x)] = 0.2722 \\
 \text{Buckling index, } \phi_b &= [S_x^2 \gamma_b / (A J)]^{0.5} = 0.4504 \\
 \text{Ratio, } \beta_w &= 1 \quad \text{for plastic section}
 \end{aligned}$$

$$\begin{aligned}
 \text{Equivalent slenderness, } \lambda_{LT} &= 2.25 (\phi_b \lambda \beta_w)^{0.5} = 10.24 \\
 P_E &= \pi^2 E / \lambda_{LT}^2 = 18824.78
 \end{aligned}$$

$$\begin{aligned}
 \alpha_{LT} &= 7 \\
 \lambda_{L0} &= 0.4 (\pi^2 E / p_y)^{0.5} = 37.89 \\
 \text{Perry factor, } \eta_{LT} &= \alpha_{LT} (\lambda_{LT} - \lambda_{L0}) / 1000 = -0.1936 < 0 \\
 \phi_{LT} &= [p_y + (\eta_{LT} + 1) P_E] / 2 = 7700.15
 \end{aligned}$$

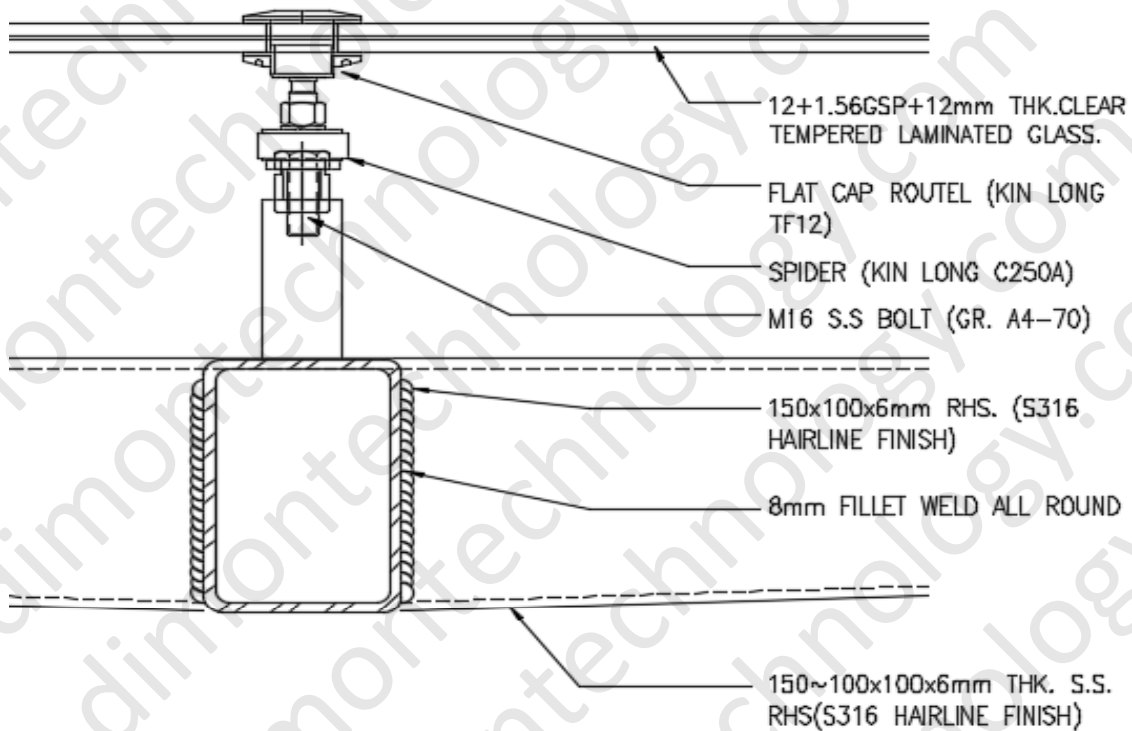
$$\begin{aligned}
 \text{Bending buckling strength, } p_b &= P_E p_y / (\phi_{LT} + (\phi_{LT}^2 - P_E p_y)^{0.5}) \\
 &= 273.79 \quad \text{N/mm}^2 \\
 &> 220 \quad \text{N/mm}^2
 \end{aligned}$$

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5.2 Check for 8mm fillet weld all round connection to horizontal member



Vertical load = 17.778 kN (refer to item 5.1)

Moment = 18.62 kNm (refer to item 5.1)

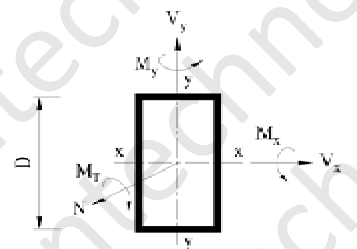
Properties for unit throat thickness of fillet weld (effective length)

Breadth,	$B = 100$	mm
Height,	$D = 150$	mm
Area,	$A = 2(B + D)$	= 500 mm ²
Moment of inertia,	$I_x = B D^2 / 2 + D^3 / 6$	= 1687500 mm ⁴
Moment of inertia,	$I_y = B^2 D / 2 + B^3 / 6$	= 916667 mm ⁴
Polar moment of inertia,	$J = I_x + I_y$	= 2604167 mm ⁴

Shear load,	$V_x = 0$	kN	Moment,	
Shear load,	$V_y = 17.778$	kN	Moment,	$M_y = 0$
Tensile load,	$N = 0$	kN	Torsional moment,	$M_T = 0$

Leg length of fillet weld, $t = 8$ mm

Shear stress,	$\tau_x = V_x / (0.7 t A) + M_T (D / 2) / (0.7 t J)$	= 0	N/mm ²
Shear stress,	$\tau_y = V_y / (0.7 t A) + M_T (B / 2) / (0.7 t J)$	= 6.35	N/mm ²
Tensile stress,	$\sigma = N / (0.7 t A)$	= 0	N/mm ²
Tensile stress,	$\sigma_x = M_x D / (2 I_x) / (0.7 t)$	= 147.78	N/mm ²
Tensile stress,	$\sigma_y = M_y B / (2 I_y) / (0.7 t)$	= 0	N/mm ²



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Resultant , $f_w = [\tau_x^2 + \tau_y^2 + (\sigma + \sigma_x + \sigma_y)^2]^{0.5}$
 $= 147.92 \text{ N/mm}^2$
 $\leq 200 \text{ N/mm}^2$ O.K.

5.3 Check for horizontal member 150x100x6mm RHS

Vertical load $= 2 \times 17.778$ (refer to item 5.1)
 $= 34.896 \text{ kN}$

Torsiona moment $= 18.62 \text{ kNm}$ (refer to item 5.2)

Distance of support $= 2325 \text{ mm}$

Reaction at support $= 34.896 / 2$
 $= 17.448 \text{ kN}$

Max bending moment $= 34.896 \times 2.325 / 4$
 $= 20.28 \text{ kNm}$

Shear stress $= 17.778 \times 1000 / (6 \times 150) + 18.62 / 147000$
 $= 19.39 \text{ N/mm}^2$
 $< 0.6 \times 127 = 76.2 \text{ N/mm}^2$ (low shear)

Moment capacity $= \min(1.2 p_y Z_y, p_y S_y)$
 $= (1.2 \times 220 \times 115/1000, 220 \times 141 / 1000)$
 $= 30.36 \text{ kNm}$ (factored)
 $> 20.28 \text{ kNm}$ O.K.

Max .deflection $= 1/48 P_k L^3 / (E I)$
 $= 34.896 \times 5.13 / 7.46 \times 1000 \times 2325^3 / (48 \times 200000 \times 8620000)$
 $= 3.645 \text{ mm}$ (unfactored)
 $< 2325 / 250 = 9.3 \text{ mm}$ O.K.

Check for lateral torsional buckling

Design strength, $p_y = 220 \text{ N/mm}^2$ Parameter, $\varepsilon = (275 / p_y)^{0.5} = 1.118$
 Modulus of Elasticity, $E = 200000 \text{ N/mm}^2$

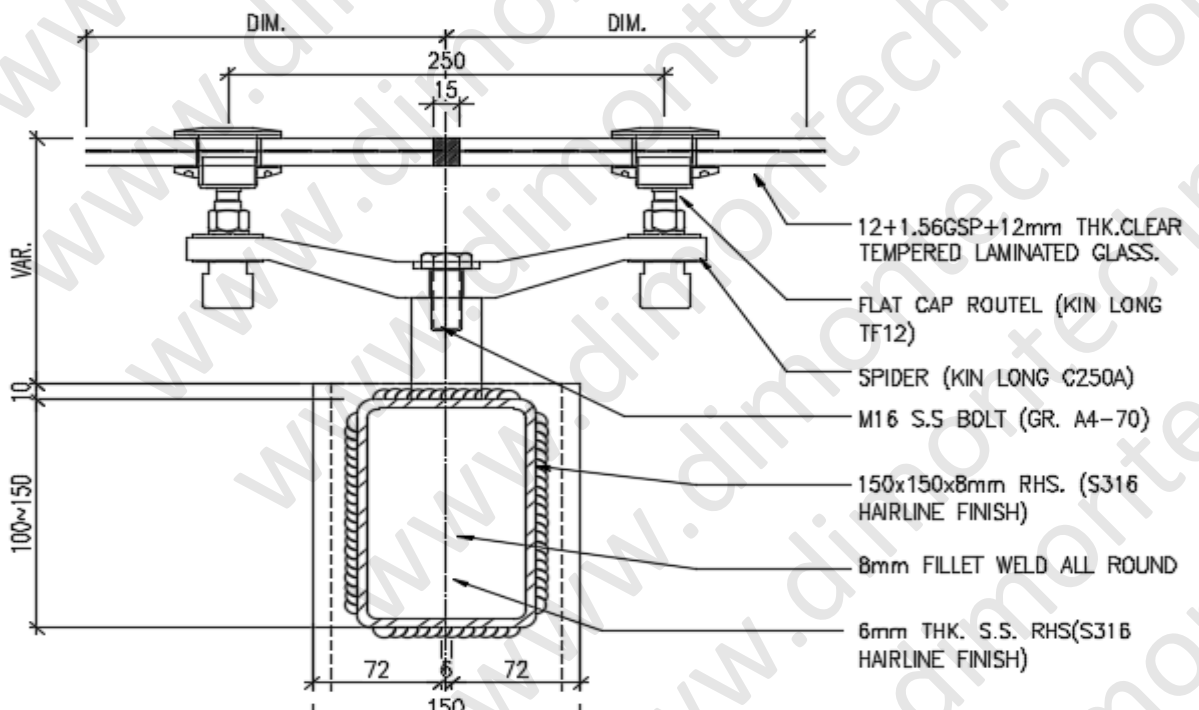
Overall width,	B = 100	mm	Overall depth,	D = 150	mm
Wall thickness,	t = 6	mm	Area,	A = 28.2	cm ²
Moment of inertia,	I _x = 862	cm ⁴	Moment of inertia,	I _y = 456	cm ⁴
Section modulus,	Z _x = 115	cm ³	Section modulus,	Z _y = 91.2	cm ³
Plastic modulus,	S _x = 141	cm ³	Plastic modulus,	S _y = 106	cm ³
Torsional constant,	J = 946	cm ⁴	Radius of gyration,	r _y = 4.02	cm

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Effective length,	$L_E = 2362$	mm	Torsional constant,	$C = 147$	cm ³
Slenderness ratio,	$\lambda = L_E / r_y$			$= 46$	
	$\gamma_b = (1 - I_y / I_x) [1 - J / (2.6 I_x)]$			$= 0.2722$	
Buckling index,	$\phi_b = [S_x^2 \gamma_b / (A J)]^{0.5}$			$= 0.4504$	
Ratio,	$\beta_w = 1$				for plastic section
Equivalent slenderness,	$\lambda_{LT} = 2.25 (\phi_b \lambda \beta_w)^{0.5}$			$= 10.24$	
	$P_E = \pi^2 E / \lambda_{LT}^2$			$= 18824.78$	
	$\alpha_{LT} = 7$				
	$\lambda_{L0} = 0.4 (\pi^2 E / p_y)^{0.5}$			$= 37.89$	
Perry factor,	$\eta_{LT} = \alpha_{LT} (\lambda_{LT} - \lambda_{L0}) / 1000$			$= -0.1936$	< 0
	$\phi_{LT} = [p_y + (\eta_{LT} + 1) P_E] / 2$			$= 7700.15$	
Bending buckling strength,	$p_b = P_E p_y / (\phi_{LT} + (\phi_{LT}^2 - P_E p_y)^{0.5})$				
	$= 273.79$	N/mm ²			
	> 220	N/mm ²			
Capacity of buckling moment	$= 220 \times 141 / 10^3$				
	$= 31.02$	kNm			(factored)
	> 18.62	kNm			O.K.

5.4 Check for 8mm fillet weld all round connection horizontal member to column



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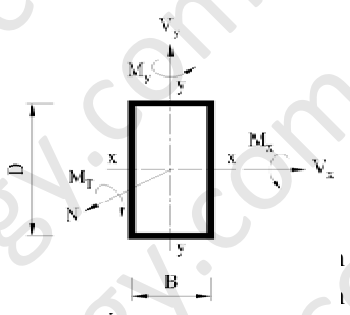
$$V = \frac{18.62}{150}$$

Torsional Moment = 18.62 / 2 = 9.31 kNm (refer to item 5.3)

Properties for unit throat thickness of fillet weld (effective length)

Breadth, $B = 100$ mm
 Height, $D = 150$ mm
 Area, $A = 2(B + D) = 500$ mm²
 Moment of inertia, $I_x = B D^2 / 2 + D^3 / 6 = 1687500$ mm⁴
 Moment of inertia, $I_y = B^2 D / 2 + B^3 / 6 = 916667$ mm⁴
 Polar moment of inertia, $J = I_x + I_y = 2604167$ mm⁴

Shear load, $V_x = 0$ kN Moment,
 Shear load, $V_y = 17.448$ kN Moment,
 Tensile load, $N = 0$ kN Torsional moment, $M_T = 9.31$ kNm



Leg length of fillet weld, $t = 8$ mm

Shear stress, $\tau_x = V_x / (0.7 t A) + M_T (D / 2) / (0.7 t J) = 47.88$ N/mm²
 Shear stress, $\tau_y = V_y / (0.7 t A) + M_T (B / 2) / (0.7 t J) = 38.15$ N/mm²
 Tensile stress, $\sigma = N / (0.7 t A) = 0$ N/mm²
 Tensile stress, $\sigma_x = M_x D / (2 I_x) / (0.7 t) = 73.89$ N/mm²
 Tensile stress, $\sigma_y = M_y B / (2 I_y) / (0.7 t) = 0$ N/mm²

Resultant, $f_w = [\tau_x^2 + \tau_y^2 + (\sigma + \sigma_x + \sigma_y)^2]^{0.5}$
 $= 95.96$ N/mm²
 ≤ 200 N/mm² O.K.

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5.5 Check for column 150x150x8mm SHS

Properties of gms. 150x150x8mm thk. SHS.

Moment of inertia,	$I = 1491$	cm^4	Area,	$A = 44.8$	cm^2
Section modulus,	$Z = 199$	cm^3	Plastic modulus,	$S = 237$	cm^3
Torsional constants of inertia,	$J = 2351$	cm^4	Torsional constants of modulus	$C = 291$	cm^3
Radius of gyration	$r = 5.77$	cm			

Vertical load = 2×17.778 (refer to item 5.1)
 = 34.896 kN

Moment = 2×18.62
 = 37.24 kNm (refer to item 5.1)

Moment capacity = $\min(1.2 p_y Z_y, p_y S_y)$
 = $(1.2 \times 220 \times 199 / 1000, 220 \times 237 / 1000)$
 = 52.14 kNm (factored)
 > 37.24 kNm O.K.

Design strength,	$p_y = 220$	N/mm^2	Robertson constant,	$\alpha = 2$	
Modulus of Elasticity,	$E = 200000$	N/mm^2	Radius, of gyration,	$r_y = 57.7$	mm
Effective length,	$L_E = 1529$	mm			

Slenderness, $\lambda = L_E / r_y = 26.4991$
 Limit slenderness, $\lambda_0 = 0.2 (\pi^2 E / p_y)^{0.5} = 18.9445$
 Perry factor, $\eta = \alpha (\lambda - \lambda_0) / 1000 = 0.0151 \geq 0$ O.K.
 $P_E = \pi^2 E / \lambda^2 = 2811.043$
 $\phi_c = [p_y + (\eta + 1) P_E] / 2 = 1536.745$

Compressive buckling strength, $p_c = P_E p_y / [\phi_c + (\phi_c^2 - P_E p_y)^{0.5}]$
 = 216.46 N/mm^2

Compression resistance $P_c = 216.46 \times 44.8 \times 100$
 = 969.741 kN
 > 34.896 kN O.K.

Combined loads = $34.896 / 969.741 + 37.24 / 52.14$
 = 0.75
 < 1 O.K.

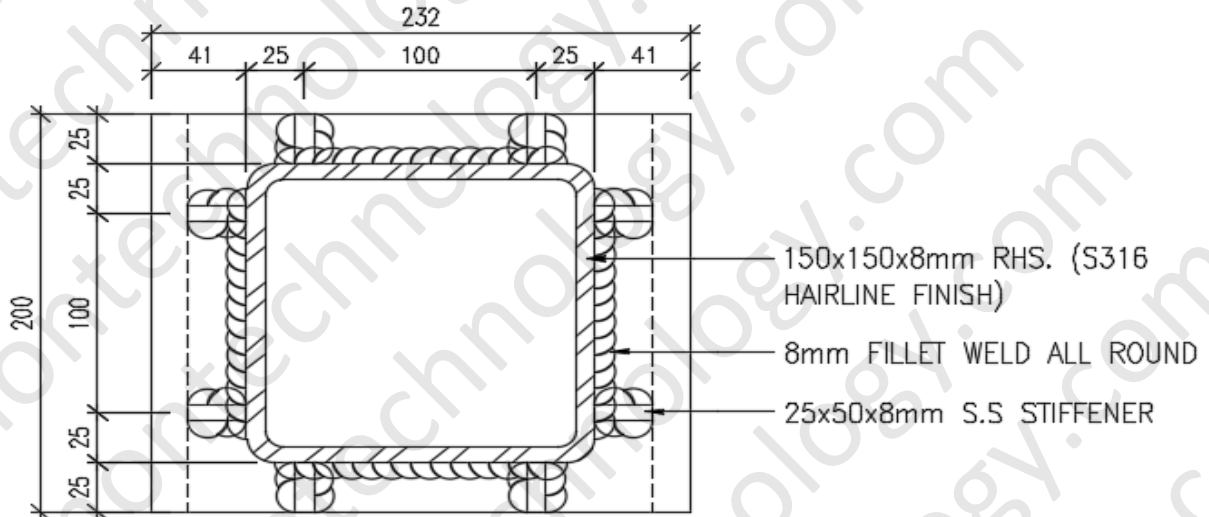
5.6 Check for 8mm fillet weld all round connection to base plate

Vertical load = 34.896 kN (refer to item 5.5)

Moment = 37.24 kNm (refer to item 5.5)

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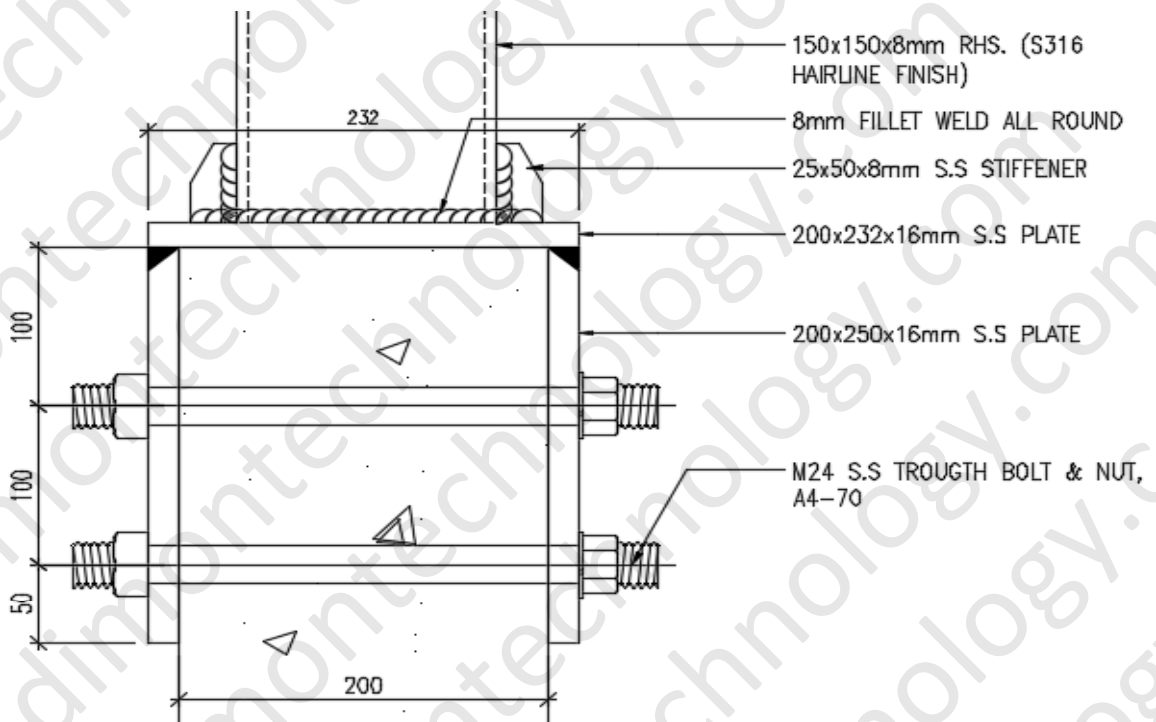


Breadth,	$B = 200$	mm		
Height,	$D = 200$	mm		
Area,	$A = 16 \times 25 + 4 \times 150$		$= 1000$	mm^2
Moment of inertia,	$I_x = B D^2 / 2 + D^3 / 6 + 8 \times 25 \times 87.5^2 + 4 \times 25 \times 54^2 + 4 \times 25 \times 46^2$		$= 4284450$	mm^4
Moment of inertia,	$I_y = B^2 D / 2 + B^3 / 6 + 8 \times 25 \times 87.5^2 + 4 \times 25 \times 54^2 + 4 \times 25 \times 46^2$		$= 4284450$	mm^4
Polar moment of inertia,	$J = I_x + I_y$		$= 8568900$	mm^4
Shear load,	$V_x = 0$	kN	Moment,	$M_x = 37.24$ kNm
Shear load,	$V_y = 0$	kN	Moment,	$M_y = 0$ kNm
Tensile load,	$N = 34.896$	kN	Torsional moment,	$M_T = 0$ kNm
Leg length of fillet weld,	$t = 8$	mm		
Shear stress,	$\tau_x = V_x / (0.7 t A) + M_T (D / 2) / (0.7 t J)$		$= 0$	N/mm^2
Shear stress,	$\tau_y = V_y / (0.7 t A) + M_T (B / 2) / (0.7 t J)$		$= 0$	N/mm^2
Tensile stress,	$\sigma = N / (0.7 t A)$		$= 6.23$	N/mm^2
Tensile stress,	$\sigma_x = M_x D / (2 I_x) / (0.7 t)$		$= 155.21$	N/mm^2
Tensile stress,	$\sigma_y = M_y B / (2 I_y) / (0.7 t)$		$= 0$	N/mm^2
Resultant ,	$f_w = [\tau_x^2 + \tau_y^2 + (\sigma + \sigma_x + \sigma_y)^2]^{0.5}$		$= 161.44$	N/mm^2
	≤ 200	N/mm^2		O.K.

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6. Check for embed



Moment = 37.24 kNm (refer to item 5.5)

Shear load on each bolt = $37.24 / 0.232 / 4 + 34.896 / 2 / 4$
= 44.49 kN

Tensile area of M24 bolt = 352 mm²

Tensile strength of M24 bolt = 373 N/mm²

Shear strength of M24 bolt = 280 N/mm²

Shear capacity of bolt = $352 \times 280 / 1000$
= 98.56 kN
> 44.49 kN

O.K.

Bearing pressure on concrete = $44.49 \times 1000 / (24 \times 100)$
= 18.538 N/mm²
< $0.6 \times 45 = 27$ N/mm²

O.K.